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ZIEGS, Carsten

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Application No. not known

Examiner: not known

Priority claimed from: Federal Republic of Germany application 201 02 076.9

Filed: February 6, 2002

For: DEVICE FOR BRAKING A MOTOR SHAFT

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February 6, 2002

CLAIM FOR PRIORITY

Hon. Commissioner of Patents
And Trademarks
Washington, D.C. 20231

Sir:

In connection with the above-identified application, applicants claims priority from Federal Republic of Germany (FRG) patent application 201 02 076.9, filed February 7, 2001.

Respectfully submitted,



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TRANSLATOR'S VERIFICATION

I, Philip M. Morris, a translator residing at P.O. Box 670907, Dallas, Texas 75367 verify that I know well both the German and the English languages, that I have prepared the attached English translation of a U.S. utility application in the German language based on FRG 201 02 076.9 and entitled "Brake Band Centering", applicant Dolmar GmbH, inventor Carsten Ziegs and that the attached English translation of this document is a true and correct translation of the documents attached thereto to the best of my knowledge and belief.

I further declare that all statements made of my own knowledge are true and that all statements made on information and belief are believed to be true, and further that these statements are made with the knowledge that willful false statements and the like are punishable by fine or imprisonment, or both, under 18 USC 1001, and that such false statements may jeopardize the validity of this document.

Date: January 31, 2001

By: Philip M. Morris

Brake Band Centering

The present invention is relative to a device for braking a motor shaft, especially for a chain saw, with a brake band that at least partially surrounds an area of the shaft.

In implements comprising rotating masses a rapid braking of these masses is often desired. For example, a drum brake consisting of brake band and coupling [clutch] drum are used to brake a chain in a chain saw. The brake bands are asymmetrically designed in this instance for an optimum utilization of material, relative to the forces occurring during braking. Furthermore, the deep-drawn coupling drums have a conicity of their outer circumference, which results in an axial force, in the interplay with the asymmetry of the braking and during the braking process, that can allow the brake band to slip off the coupling drum.

In order to secure the brake band from slipping off axially during the engagement of the brake, e.g., the area around the coupling drum is very tightly closed by an additional housing covering so that the brake band is axially secured in the braking state. A very precise positioning of the parts to each other is necessary in order to assure such a securing. In addition, a brake band heated by the braking process can damage or melt plastic coverings upon making contact with them.

In other known solutions tongues [flaps] are welded onto the brake band that are guided in radial offsets (relative to the axis of rotation of the

coupling shaft) of the component guiding the brake band, e.g., a crankcase. This means additional components, sources of errors in the production and therefore increased production costs.

The invention therefore has the basic problem of providing a device for braking a motor shaft with a brake band which device makes possible an exact guidance of the brake band without additional, separate components on the brake band and the surrounding housing.

The invention solves this problem in that the brake band comprises at least one projection designed in one piece with the brake band. It is possible, due to the features of the invention, that an exact guidance of the brake band is assured without additional components on the brake band or the housing, which eliminates additional costs for components that hold the brake band in position.

According to a preferred embodiment of the invention the at least one projection is designed as a bulge [convexity] of the brake band in order to achieve a simplified manufacture of the brake band.

According to an especially advantageous embodiment of the invention the outside diameter of the projection is greater than the sum of the outside diameter of a loop formed by the brake band plus the radial work path of the loop in order to assure that the brake band is securely guided in its initial position as well as in the position during the braking procedure.

The guide component that holds the brake band in its axial position is preferably located on a housing.

According to an advantageous embodiment this guide component is designed in one piece with the housing in order to use already present elements of the housing for guiding and to make a simplified method of manufacture possible and save assembly work.

According to a preferred embodiment of the invention the housing comprises radial recesses for receiving the projections. The projection is preferably formed thereby by two guide components that are preferably a part of the housing in order to reduce manufacturing accosts.

According to a special embodiment of the invention the recesses are formed in the housing by counterdipping [counter-, opposite dips] in the die-casting tool in order to make possible a simple, economical mass production.

An exemplary embodiment of the invention is explained below with reference made to the drawings and described using the example of a chain saw.

Figure 1 shows a perspective side view of a chain saw with motor shaft in accordance with the present invention.

Figure 2 shows a view of figure 1 in which the coupling drum has been removed.

Figure 3 shows a perspective view of a brake band in accordance with the invention.

Figure 4 shows a detail view of brake band of the invention according to figure 3.

Figure 5 shows a detail view of figure 2 without brake band.

Figure 6 shows the detail view of figure 5 with inserted brake band.

Figure 7 shows another detail view of figure 2 without brake band.

Figure 8 shows the detail view of figure 7 with inserted brake band.

A device for braking a motor shaft in a chain saw 10 is described with reference made to figure 1. Crankcase 11 is shown in which motor shaft 12 is supported and driven by a motor (not shown). Motor shaft 12 is coupled by coupling [clutch] 13 to gear 14 that rotates about the same axis of rotation 15 as motor shaft 12. The gear serves to drive the chain (not shown) of chain saw 10.

In addition, chain guide sheet 16 is attached to crankcase 11 and assures the guidance of the chain to gear 14. The coupling comprises circular coupling drum 17 rigidly connected to gear 14 for a rotary movement about axis 15. Furthermore, circularly arranged brake band 18 lies around coupling drum 17 and is anchored on one side in crankcase 11. In order to brake the movement of the chain the rotation of coupling drum 17 is braked. To this end a tractive force is exerted on one end of brake band 18 in order to reduce the diameter of circularly arranged brake band 18. As a result thereof, brake band 18 engages with the periphery of coupling drum 17 and exerts a frictional force on drum 17 in order to brake its rotation. The periphery of the drum has a conical surface 19 on which brake band 18 comes to rest during the braking process. An exact guidance of

brake band 18 is necessary in order to unambiguously position the band when the brake is not in use and in order to prevent brake band 18 from slipping off from coupling drum 17 during the braking process.

Figure 2 shows crankcase 11 of figure 1; however, coupling drum 17 has been removed here along with the gear and the coupling in order to show the positioning of brake band 18. The support and guidance of brake band 18 is assured by at least one projection, in the present instance two projections 20, 21 located on brake band 18. Projections 20, 21 engage with guide components 22, 23 located on crankcase 11 and on the end of chain guidance sheet 24 facing the motor shaft.

Brake band 18 will first be described with reference made to figures 3, 4. Brake band 18 comprises a first end 25 and a second end 26. First end 25 is fastened to crankcase 11 (see figure 2) and second end 26 is movably arranged in crankcase 11. As previously stated, brake band 18 forms a circular loop surrounding coupling drum 17 in the assembled state. Upon an appropriate movement of second end 26 the diameter of the circular loop becomes smaller, as a result of which brake band 18 comes to contact coupling drum 17.

Brake band 18 has two projections 20, 21 for guiding brake band 18 that are in one piece with brake band 18. In the preferred exemplary embodiment of the invention projections 20, 21 are realized in the form of bulges of brake band 18. Projections 20, 21 are manufactured by deformation processes such as, e.g., stamping [impressing] on brake band

18. In particular, projections 20, 21 can be stamped as symmetric partial supports [rim holes, eyelets]. The outside diameter of the projections is greater than the sum of the outside diameter of the loop formed by mounted brake band 18 plus the radial work path of brake band 18. This assures that projections 20, 21 do not come out of engagement during the braking process with guide components 22, 23 described relating to the later figures. The guiding and holding of brake band 18 in its initial position as well as during the braking process is given therewith. Projections 20, 21 extend radially outward in relation to the axis of rotation of the motor shaft.

Figure 5 shows a detail view of the lower brake band insert area in crankcase 11 of figure 2. The brake band has been removed from the drawing. At least one of the guide components 22 is designed as recess 26 in the inner wall of the crankcase. Recess 26 can be formed in the inner wall by subsequent mechanical working or also without subsequent working by counter-dipping [counter-, opposite dips] in the die-casting tool. The recess is defined by three guide parts 27, 28, 29 comprising lateral surfaces 27a, 28a, 29a arranged normally to axis of rotation 15.

In figure 6 recess 26 is designed in such a manner that it can receive projections 20, 21 of brake band 18, during which lateral surfaces 27a, 28a, 29a engage with projections 20, 21 in order to guide brake band 18 in its radial movement from the initial position into the position during the braking process. An unintended axial slipping of brake band 18 is thus prevented by an embedment [inclusion] of projections 20, 21 of the brake band in recess

26. The dimensions of projections 20, 21 are selected so that even in the braking position projections 20, 21 remain embedded [included] in recess 26.

Figures 7 and 8 show another detail view of crankcase 11 and comprise another guidance, in accordance with the invention, of brake band 18. A contact surface 30 of crankcase 11 defines together with zone of contact 31 of the chain guide sheet a guidance recess or a guidance conduit. A projection 20, 21 of brake band 18 is received in this guidance conduit 32 and guides brake band 18 in its movement and prevents an axial slipping of brake band 18 in the initial position and in the braking position.

Even though the device was described using a chainsaw 10, a device designed in accordance with the invention can also be used in other implements and machines in which a motor shaft is braked by a brake band without departing from the concept of the invention.

CLAIMS:

1. A device for braking a motor shaft (12), especially for a chain saw, with a brake band (18) that at least partially surrounds an area of the shaft (12) and with at least one guide component (22, 23) that cooperates with it [the brake band] and holds the brake band (18) in its axial position, characterized in that the brake band (18) comprises at least one projection (20, 21) formed in one piece with the brake band.
2. The device according to claim 1, characterized in that the projection (20, 21) is a bulge of the brake band (18).
3. The device according to claim 1 or 2, characterized in that the outside diameter of the projections (20, 21) is greater than the sum of the outside diameter of a loop (18) formed by the brake band plus the radial work path of the loop.
4. The device according to claims 1 to 3, characterized in that the guide component (22, 23) is attached to the housing (11).
5. The device according to claims 1 to 4, characterized in that the guide component (22, 23) is formed in one piece with the housing (11).
6. The device according to claims 1 to 5, characterized in that the housing (11) comprises at least one radial recess (26, 32) for receiving the projections (20, 21).
7. The device according to claims 1 to 6, characterized in that the recess (26, 32) is formed by two guide components (30, 31).

8. The device according to claims 1 to 7, characterized in that the recess (26, 32) is formed in the housing (11) by counterdipping [counter-, opposite dips] in the die-casting tool.

ABSTRACT:

The invention is relative to a device for braking a motor shaft (12), especially for a chain saw, with a brake band (18) that at least partially surrounds an area of the shaft (12) and with at least one guide component (22, 23) that cooperates with it [the brake band] and holds the brake band (18) in its axial position, which brake band (18) comprises at least one projection (20, 21) formed in one piece with the brake band.